



METHOD FOR EVALUATING THE DURABILITY OF THE ROLLING MILLS

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ABSTRACT:

The researches on the durability in exploitation of hot rolling mill cylinders represent an important scientific and economical issue. The study represents a detailed approach of the influence of various technological factors on the durability in exploitation of rolling mill cylinders made of different steel and pig iron grades and suggest solutions meant to increase the durability of the rolls in exploitation. The purpose of this work is to increasing of durability and safety in operation.

KEYWORDS:

rolls, durability, fatigue, cycles, steel, iron

1. INTRODUCTION

The classic methods for evaluating the durability of cylinders do not provide an answer to many of the phenomena accruing in the rolling process, between cylinders and laminate. These methods do not take into consideration the highly important thermal influences, which constitute one of the fundamental causes leading to the destruction of hot rolling mill cylinders and, also, may reach considerable values that can be observed only through experiments. The lack of detailed, theoretical and experimental research upon thermo – mechanical processes that take place during plastic deformation in rolling cylinders, constitutes a factor, which reduces the possibilities of rational exploitation in rolling mills.

The cylinders are apply to thermal tensions who are variable, complex, with extremely marked influences. Therefore, to intensify the rolling processes we need to observe the durability limits, with thermal tensions produced in symmetrical and asymmetrical temperature fields, at a large number of stress cycles. To this purpose it is necessary to know as accurately as possible the type of stress, the materials, and a detailed characterises evaluation, to determine exploitation timing and to compare with previously established values. The study representes a detailed approach of the influence of various technological factors on the durability in exploitation of rolling mill cylinders made of different steel and pig iron grades and suggests solutions meant to increase the hardness of rolling mill cylinders in exploitation.

2. THE EXPERIMENTAL DURABILITY RESEARCHES

Hardness testing is done on a series of rings, achieved from the cylinders' axles resulted from the industrial operation and which accomplished the rolling drives. In experiments these rings are subject to different conditions of cyclic thermal requirements, that during a rotation, they turn to heat in a furnace containing electric resistors, at different imposed temperature on the one hand and on the other hand, they turn cold in different environments: air (A regime), water (B regime) and carbonic snow jets (C regime), [1].

In the experimentally installation for the research on the durability in exploitation of the steel and iron marks, until the appearance of the thermal fatigue cracks is presented. With a view to choosing the materials on which hardness testing is about to be done, and respectively the types of steel and cast iron frequently used at the achievement of rolling cylinders, it was necessary the studying of constructive parameters and the features on the behavior in operation of all cylinders found in the industrial housing of a rolling-mill from an iron and steel combine, which has a rolling capacity of over 2 million tons of steel per year. As to the types of steel and cast iron used at the achievement of cylinders, these are limited and refer to six qualities (three steel and three cast iron) as well as: 65VMoCr15 – steel used to manufacture rolls from semi-finished mills; 55VMoCr12 – steel used to manufacture rolls from heavy section mills; OTA3 - Steel used to manufacture rolls from heavy, medium and light mills; FNS 2 – iron used in the making of rolls in heavy section mills; FD2 - iron used in the making of rolls and light section and wire mills.

From the study of the durability of the hot rolling cylinders, where have been registred isochronal diagrams representing the rolling cylinders temperature variations, coresponding to a two circular measure angle, resulted that the maximal asymmetrical temperatures appears at diminishing rolling speed, respectively diminishing rotation numbers of the rolls, [2]. Once with the increment of the rotation numbers of the rolls, at high rolling speeds decrease the asymmetric temperature fields so, decrese also the calibers thermal fatigue loading. On these bases, we chose the minimal value for the rotation number of the tryouts constrained durability test being as 35,7 rot/min, producing the highest thermal fatigue because the thermal tension appearing as effect of temperature variations are maximal and after a relative small number of rotations, appear the first thermal fatigue cracks.

In order to increase the number of the loading cycles, until the first thermal fatigue cracks appear, we have tried to maintain as high as possible temperature for tryouts and the cooling fast and accentuated. Each of the three sets of tryouts consisting in six rings were constrained to a working regime, pursuing the calculated moment of the appearance of the thermal fatigue first cracks, registering the number of loading cycles. In this way has been registred the cyclic temperature variations in points, at the surface ($\Delta r = 0$) and in the superficial layer ($\Delta r = 1,5$ and 3,0 mm). The synthesis of the characteristic data for the registered temperature variations in the experimental loading regime A, B and C are presented in the table 1.

In stress regime A, the materials under study resisted longest at stress cycles, untill the first thermal fatigue cracks appeared, and this regime is called loading regime. In stress regime B, the first thermal fatigue cracks appeared in a smaller number of stress cycles and this is a medium regime. In stress regime C, the thermal fatigue cracks appeared at the lowest number of stress cycles and it is called a heavy regime. The durability histograms for the steel and cast iron marks are presented in fig.1, 2 and 3.

Depth of the superficial	The diagram of cyclic temperature variations in working regime A		The diagram of cyclic temperature variations in working regime B		The diagram of cyclic temperature variations in working regime C	
layer Δ r	r The determinate tem			nperature vari	ation [ºC]	
[mm]	Maximal	Minimal	Maximal	Minimal	Maximal	Minimal
0	809,7	265,2	796,4	203,1	766,3	197,1
1,5	639,5	250,4	543,7	193,6	504,6	175,3
3 ,0	508,1	233,5	338,3	162,7	298,4	138,8
Number of rotations n = 35,7 rot/min $\phi_i = \phi_e = 180^\circ = 2\pi$ rad						

TABLE1. The minimal and maximal level for cyclical variation of temperature after the isocronal diagrams



FIGURE 1. Durability histograms for "A" thermal regime



FIGURE 2. Durability histograms for "B" thermal regime



FIGURE 3. Durability histograms for "C" thermal regime

After analize the durabity histograms the synthesis for number of thermal loading cycles aiming by the appearance of the first thermal fatigue cracks is presented in table 2.

TABLE 2.	The synthesis for number	of thermal	loading cycles	aiming by the	appearance	of the
		first therm	al fatigue crack	S		

The steel and		Number of thermal loading cycles			
No.	the cast iron marks	Working regimes			
		А	В	С	
1.	55 V MoCr 12	5265	4005	3647	
2.	90 V MoCr15	9121	7954	7110	
3.	OTA 3	18023	16122	14865	
4.	65 V Mo Cr 15	7068	6021	5206	
5.	FNS 2	10642	9328	8688	
6.	FD 2	9078	8620	7469	

If analyse table 2 we can see that the type of stress which gave the best results regarding stability to thermal fatigue - studied in stress regime C – is the OTA 3 steel type, underwent to 14865 cycles until the first cracks of thermal fatigue. Other types of steel 55VMoCr12 and 90VMoCr15 underewent relatively well to the stress of thermal fatigue in stress regime A and acceptably well in stress regimes B and C. In the case of the two types of iron used in experimental research a better behavior was noticed at FNS 2 which underwent to 8688 cycles in stress regime C, until the first cracks of thermal fatigue. Iron type FD2 behaves acceptably and is used to produce hard rolls from finishing stands.

3. RESULTATES

In order to evaluate the results of the researches upon the experimental rings durabilities to thermal fatigue and the extension of comparative results with the durabilities cylinders exploitation from the industrial bar mills, is imperative to determine the temperature fields variations also in un-dimensional form. So, the temperature field's variations during both the experimental rolling, and during the durabilities experimentations of the rings, will be calculated in specific undimensional temperature variations, which is determined with relation (1).

$$\theta = \frac{t - t_{\min}}{t_{\max} - t_{\min}} \tag{1}$$

In relation (1): θ – specific temperature, it is a parameter, it's values are between 0 and 1; t – variable temperature in the cylinder; t_{min} – minimum temperature; t_{max} – maximum temperature;

If the variable temperature in the cylinder is $t = t_{min}$, in this conditions $\theta = 0$ and for $t = t_{max}$, $\theta = 1$.

The dimensionless radius in the quaquaversal section of the cylinders is determined with relation (2).

$$p = \frac{r}{R}$$
(2)

In relation (2): ρ – dimensionless radius in the quaquaversal section of the cylinders which change in 0 and 1; r - mensurate radius in the sectiunea quaquaversal section of the cylinders; R – maximum radius in the quaquaversal section of the cylinders. For r = 0, ρ = 0 and if r = R, ρ = 1.

The medium values of the temperature variations $\overline{\theta}$ will determining with Simpson's as well as represent the integral of the function that describes the temperature field.

The use of the specific temperature variations and the specific radius in undimensional form is necessary because the temperature variations in the cylinders are smaller in comparison to the thermal variations from the experimental rings. These variations were intentionally increased in order to decrease the experimental time until the appearance of the first cracks produced by the overload of the thermal regimes.

In this sense, the number of rings real thermal fatigue solicitation cycles would have been bigger in the case of similar solicitations then in the case of the experimental rolling. So, in order to establish the real number of solicitation cycles, it is imperative to calculate the thermal coefficient K_T , which is determined with relation (3).

$$\mathbf{K}_{\mathrm{T}} = \frac{\overline{\overline{\theta}}_{\mathrm{exp\,erimentareprobe-ineleladurabilitate}}}{\overline{\overline{\theta}}_{\mathrm{la\,min\,are\,exp\,erimentala}}}$$
(3)

In which: $\overline{\overline{\theta}}_{experimentalrolling}$ - mean specifically temperature, middle of the three variations level of temperature ($\Delta r=0$; 1,5;3 mm) from the experimental rolling diagram with n=35,7rot/min; $\overline{\overline{\theta}}_{experimentalrings}$ - mean specifically temperature, middle of the temperature variations from the diagram of rings experimental thermal solicitation.

The synthesis of the temperature variation registred to the experimental rolling with n = 35.7 rot/min is presented in table 3.

Through similar calculus (presented in Table 3) are determined the specific temperature, mean of middle values $\overline{\overline{\theta}}$, after the processing the diagrams corresponding the experimental regimes A, B and C, with n = 35,7 rot/min. The results are presented in table 4, among the values of the thermal coefficient, K_T.

The number of thermal solicitation cycles in the regimes A, B and C, is smaller then the real number that would have been obtained if the loading temperature were identical with the experimental rolling temperature.

Depth of		Determinate temperature					
the Particular		Maximum		Medium		Minimum	
superficial	radius						
layer Δ r [mm]	ρ	[°C]	θ_{max}	<i>ī</i> [°C]	$\overline{ heta}$	[°C]	θ_{min}
0	1	524,8	1	169,5	0,29738	128,5	0
1,5	0,9923469	362,9	1	172,4	0,30320	126,7	0
3,0	0,9846938	211,6	1	115,33	0,18978	84,0	0
\vec{t} , $\vec{\theta}$ - average temperature			152,41	0,263453	-	-	

TABLE 3. The synthesis of the temperature variation registred to the experimental rolling with n = 35,7 rot/min

In order to determine the real solicitation cycles number until the appearance the first thermal fatigue cracks, we have to multiply the experimental cycles number with the values of the thermal coefficient, K_T , obtained through calculus for each experimental regime, table 4.

TABLE 4. The determination of the thermal influence coeficient K_T for the level of the cyclical variations of temperature under three working regimes

The specific average temperature, average from the averages calculated to the					
expe	erimental rolling $\overline{ heta}_{la \min are} = 0,26$	53453			
The values of the specific average temperature, $\bar{\theta}$, calculated for the A, B, C working regimes					
Working regime A	Working regime A Working regime B Working regime C				
$\bar{\theta}_{A} = 0,556759$	$\stackrel{=}{\theta}_{\rm B} = 0,529776$	$\bar{\theta}_{\rm C} = 0,511069$			
Thermal influence coeficient					
$K_{TA} = \frac{0,556759}{0,263453} = 2,11331 \qquad K_{TB} = \frac{0,529776}{0,263453} = 2,01089 \qquad K_{TC} = \frac{0,511069}{0,263453} = 1,93988$					

TABLE 5. The real solicitation cycles number until the appearance the first thermal fatigue cracks in the works regime A,B and C calculated with the values of the thermal coefficient, K_T

	The steel and cast iron marks	The real solicitation cycles number until the appearance the first thermal fatigue cracks				
No.		Working regime A	Working regime B	Working regime C		
		The values of the thermal influence coeficient				
		K _{TA} = 2,11331	К _{тв} = 2,01089	K _{TC} = 1,93988		
1.	55VMoCr12	11118,1	8053,6	7074,7		
2.	90VMoCr15	19275,5	15994,6	13792,5		
3.	OTA3	38088,1	32419,5	28836,3		
4.	65VMoCr15	14936,8	12107,5	10099,0		
5.	FNS2	22451,8	18757,5	16853,6		
6.	FD2	19184,6	17333,8	14488,9		

The real solicitation cycles number until the appearance the first thermal fatigue cracks in the works regime A, B, C calculated with the values of the thermal coefficient, K_T .

To compare the rings durabilities, expressed in number of thermal cycles, with the durabilities of the industrial exploitation of cylinders, we have to compare each analyzed type of materials (steels and irons). Therefore, we transform the quantity of rolled materials expressed in product-tones on caliber into linear meter of finite products, and through there division at cylinders caliber circumference, we obtain the number of thermal solicitation cycles of the industrial rolls.

The mean values of durabilities in the industrial exploitation, expressed in thermal solicitation cycles, are presented in table 6, comparative with the real values of durabilities, obtained in the research of thermal solicitations of the representative types steel and iron, used in the hot rolling cylinders making process.

TABLE 6. The mean values of durabilities in the industrial exploitation, comparative with the real values of durabilities, obtained in the research of thermal solicitations of the representative types steel and iron, used in the hot rolling cylinders making process

No.	The steel and cast iron marks	The medium number of thermal loading cycles for the experimental sample after the working regimes A, B şi C	The medium number of thermal loading cycles for the industrial rolling mills
1.	55VMoCr12	8748,8	6120,61
2.	90VMoCr15	16354,2	10881,02
3.	OTA 3	33114,6	22428,15
4.	FNS 2	19354,0	4721,85
5.	FD 2	17002,2	25654,51

4. CONCLUSIONS

Analyzing the results in the previous paragraph, one can notice that:

- for the steel grades under consideration, ie. 55VMoCr12, 90VMoCr15 and OTA 3, the hardness values are relatively similar.
- under industrial work conditions, hardness values are smaller than in laboratory, because of the influence of the mechanical effects, such as friction between grooves during the rolling process.
- for steel grade 65VMoCr15 industrial hardness has not been studied, as this steel grade is used for the manufacturing of rolling cylinders used in the semi-finished parts rolling train, which at present is out of use and in conservation.
- the mean hardness of cylinders made of cast iron FNS 2 is, under industrial exploitation, 9.09 times smaller than the mean hardness obtained experimentally on the samples tried for thermal fatigue. In this situation, we had to analyze the quality of cylinders made of this grade of cast iron and we discovered that most of them had manufacturing fails, among which excessive porosities and faulty structures. In this sense, one can notice the fact that porosities have a significant influence on cylinder hardness under industrial rolling conditions, but they have no influence whatsoever upon the hardness of experimental samples.
- the hard cast iron FD 2, used in manufacturing rolling cylinders for the intermediary and finishing train of the small section rolling train, gave very good results under industrial exploitation, its mean hardness being 1.5088 times higher than on experimental sample rings.

The differences of mean hardness are fully justified, as we have determined the experimental values of hardness under work conditions A, B and C at a rotation speed of 35,7 rot/min, and the industrial cylinders of the intermediary and finishing stands of the small section rolling mill had a rotation speed of 266 ... 575 rot/min, which corresponds to a speed coefficient $K_v = 6,33$... 16,106 that increases the number of thermal fatigue stress cycles up to the appearance of the first specific fissures.

In this situation, we can conclude that for relatively small rolling rates, temperature variations are higher and the phenomenon of thermal fatigue is more accentuated. At high rolling rates, where the number of rotations is significant, reaching in this case la 266...575 rot/min, temperature variations are small and thermal fatigue is dimmed, while the number of thermal fatigue stress cycles increases up to the appearance of the first fissures, specific to this phenomenon.

The results of these researches allow an extension of the studies to other steel and cast iron grades that show a higher strength and stability to thermal fatigue, which offers rolling cylinders a higher endurance in exploitation.

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